

ASSIGNMENT - 02

Name - Jyod Mohammad
Roll no - CSJMA20001890298
Branch - Mechanical Engineering

Submitted by
Jyod Mohammad

Submitted to
Respected Er: Ranbir
Mukhtya

Cauchy's Law -

Problem-01 - Solution

(a) For traction vector using Cauchy's law

$$\begin{bmatrix} t_1^{(n)} \\ t_2^{(n)} \\ t_3^{(n)} \end{bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{21} & \sigma_{31} \\ \sigma_{12} & \sigma_{22} & \sigma_{32} \\ \sigma_{13} & \sigma_{23} & \sigma_{33} \end{bmatrix} \begin{bmatrix} n_1 \\ n_2 \\ n_3 \end{bmatrix}$$

$$= \frac{1}{\sqrt{3}} \begin{bmatrix} 2 & 1 & 2 \\ 1 & 0 & -1 \\ 2 & -1 & 2 \end{bmatrix} \begin{bmatrix} 1 \\ 1 \\ 1 \end{bmatrix}$$

$$= \frac{1}{\sqrt{3}} \begin{bmatrix} (2 \times 1 + 1 \times 1 + 2 \times 1) \\ (1 \times 1 + 0 \times 1 + -1 \times 1) \\ (2 \times 1 + (-1) \times 1 + 2 \times 1) \end{bmatrix}$$

$$\begin{bmatrix} t_1^n \\ t_2^n \\ t_3^n \end{bmatrix} = \frac{1}{\sqrt{3}} \begin{bmatrix} 5 \\ 0 \\ 3 \end{bmatrix}$$

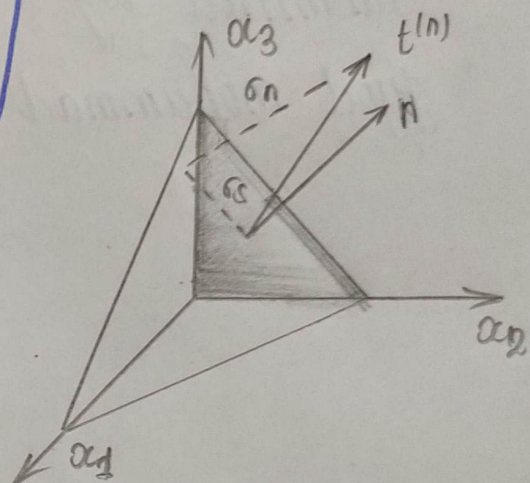
so that traction vector -

$$t^{(n)} = \left(\frac{5}{\sqrt{3}} \hat{e}_1 + 0 + \sqrt{3} \cdot \hat{e}_3 \right)$$

The component normal to the plane is

$$\begin{aligned} \sigma_n &= (t)^{(n)} \cdot n \\ &= \left(\frac{5}{\sqrt{3}} \hat{e}_1 + \sqrt{3} \cdot \hat{e}_3 \right) \cdot \left(\frac{\hat{e}_1 + \hat{e}_2 + \hat{e}_3}{\sqrt{3}} \right) \end{aligned}$$

$$\sigma_n = \left(\frac{5}{3} + 0 + 1 \right) \Rightarrow \sigma_n = \frac{8}{3} = 2.666$$



The Shearing Component of the traction is -

$$\sigma_s = \sqrt{(t_n)^2 - \sigma_n^2}$$

$$= \sqrt{\left[\left(\frac{5}{\sqrt{3}}\right)^2 + (\sqrt{3})^2\right] - \left(\frac{8}{3}\right)^2}$$

$$= \sqrt{(9.333 - 7.111)}$$

$$\sigma_s = 1.490$$

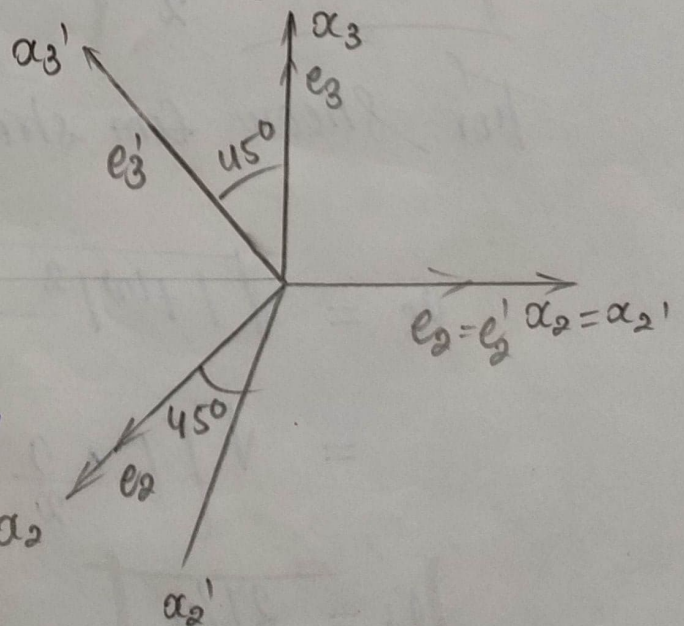
Problem-02

Two different co-ordination Systems at a

Point —

From the Cauchy's law,
the traction vector is

$$\begin{bmatrix} t_1^{(n)} \\ t_2^{(n)} \\ t_3^{(n)} \end{bmatrix} = \begin{bmatrix} 1 & 3 & 2 \\ 3 & 1 & 0 \\ 2 & 0 & -2 \end{bmatrix} \begin{bmatrix} 1/\sqrt{2} \\ 0 \\ 1/\sqrt{2} \end{bmatrix} \alpha_2$$



$$\begin{bmatrix} t_1^{(n)} \\ t_2^{(n)} \\ t_3^{(n)} \end{bmatrix} = \begin{bmatrix} (1 \times \frac{1}{\sqrt{2}} + 3 \times 0 + 2 \times \frac{1}{\sqrt{2}}) \\ (3 \times \frac{1}{\sqrt{2}} + 1 \times 0 + 0 \times \frac{1}{\sqrt{2}}) \\ (2 \times \frac{1}{\sqrt{2}} + 0 \times 0 + (-2) \times \frac{1}{\sqrt{2}}) \end{bmatrix}$$

$$= \begin{bmatrix} 3/\sqrt{2} \\ 3/\sqrt{2} \\ 0 \end{bmatrix}$$

$$t^{(n)} = 3/\sqrt{2} \hat{e}_1 + \frac{3}{\sqrt{2}} \hat{e}_2 + 0 \cdot \hat{e}_3$$

normal stress component -

$$\sigma_N = t^{(n)} \cdot n$$

$$= \left(\left(\frac{3}{\sqrt{2}} \right) \hat{e}_1 + \frac{3}{\sqrt{2}} \hat{e}_2 \right) \cdot \left(\frac{1}{\sqrt{2}} \hat{e}_1 + \frac{1}{\sqrt{2}} \hat{e}_3 \right)$$

$$\boxed{\sigma_N = \frac{3}{2}}$$

For shear stress component -

$$\sigma_S = \sqrt{|t^{(n)}|^2 - \sigma_N^2}$$

$$= \sqrt{\left[\left(\frac{9}{2} + \frac{9}{2} \right) - \frac{9}{4} \right]}$$

$$\boxed{\sigma_S = \frac{27}{4}}$$

The normal to the plane is equal to e_3'
 and so σ_N should be the same as σ_{33}'
 and it is. The stress σ_3 should be
 equal to $\sqrt{(\sigma_{31}')^2 + (\sigma_{32}')^2}$ and it is. The
 results are

traction is represent in different ways,
 with components $(t_1^{(n)}, t_2^{(n)}, t_3^{(n)})$ and
 $(\sigma_{31}', \sigma_{32}', \sigma_{33}')$

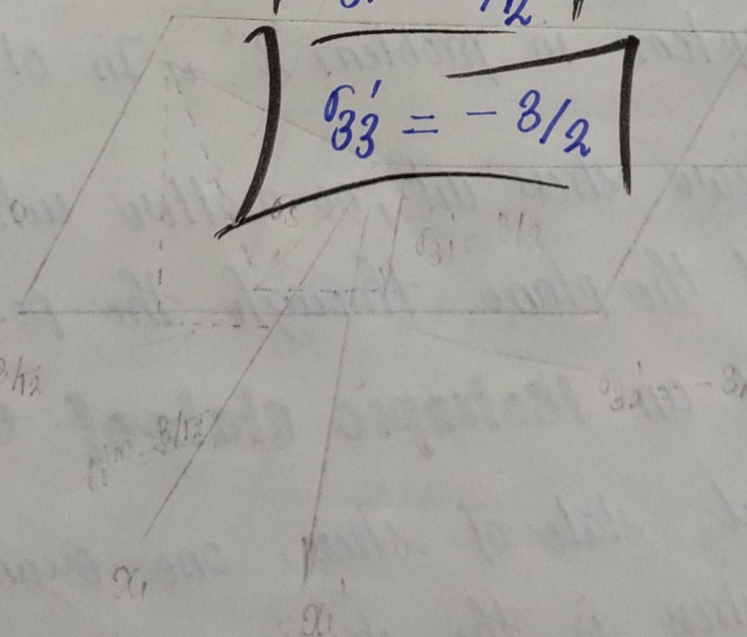
$$\boxed{\sigma_N = \sigma_{33}' = 8/2}$$

$$\boxed{\sigma_{31}' = 1/2}$$

$$\boxed{\sigma_{32}' = -3/2}$$

$$t_2^{(n)} = 3/2$$

$$t_1^{(n)} = 1/2$$



Problem - 3 - Solution -

• Isotropic state of stress

Suppose the state of stress in a body is

$$\sigma_{ij} = \sigma_0 \delta_{ij}$$

$$[\sigma] = \begin{bmatrix} \sigma_0 & 0 & 0 \\ 0 & \sigma_0 & 0 \\ 0 & 0 & \sigma_0 \end{bmatrix}$$

One finds that the application of the stress tensor transformation rule yields the very same components no matter what the new co-ordinate system in problem '2'. In other words, no shear stress act, no matter what the orientation of the plane through the point. This is termed an **isotropic state of stress**, or a spherical state of stress, one example of isotropic stress is the stressing arising in a fluid at rest, which cannot support shear stress, in which case \rightarrow

$$[\sigma] = -p[I]$$

where the scalar p is the fluid hydrostatic pressure. For this reason, an isotropic state of stress is also referred to as a hydrostatic state of stress.

• Problem - 04 - Solution -

• The Stress Tensor

Given,

$$\begin{bmatrix} \sigma_{11}' & \sigma_{12}' \\ \sigma_{21}' & \sigma_{22}' \end{bmatrix} = \begin{bmatrix} \cos\theta & \sin\theta \\ -\sin\theta & \cos\theta \end{bmatrix} \begin{bmatrix} \sigma_{11} & \sigma_{12} \\ \sigma_{21} & \sigma_{22} \end{bmatrix} \begin{bmatrix} \cos\theta & -\sin\theta \\ \sin\theta & \cos\theta \end{bmatrix}$$

$$= \begin{bmatrix} (\cos\theta \cdot \sigma_{11} + \sin\theta \cdot \sigma_{21}) & (\cos\theta \cdot \sigma_{12} + \sin\theta \cdot \sigma_{22}) \\ (\sin\theta \cdot \sigma_{11} + \cos\theta \cdot \sigma_{21}) & (-\sin\theta \cdot \sigma_{12} + \cos\theta \cdot \sigma_{22}) \end{bmatrix} \begin{bmatrix} \cos\theta & -\sin\theta \\ \sin\theta & \cos\theta \end{bmatrix}$$

$$= \begin{bmatrix} \text{PTO} \end{bmatrix}$$

$$= \begin{bmatrix} (\cos\theta \cdot \sigma_{11} + \sin\theta \cdot \sigma_{21}) \cos\theta + (\cos\theta \cdot \sigma_{12} + \sin\theta \cdot \sigma_{22}) \sin\theta \\ (-\sin\theta \cdot \sigma_{11} + \cos\theta \cdot \sigma_{21}) \cos\theta + (-\sin\theta \cdot \sigma_{12} + \cos\theta \cdot \sigma_{22}) \sin\theta \end{bmatrix}$$

$$(\cos\theta \cdot \sigma_{11} + \sin\theta \cdot \sigma_{21}) \times \cos\theta + (\cos\theta \cdot \sigma_{12} + \sin\theta \cdot \sigma_{22}) \times \sin\theta$$

$$(-\sin\theta \cdot \sigma_{11} + \cos\theta \cdot \sigma_{21}) \times -\sin\theta + (-\sin\theta \cdot \sigma_{12} + \cos\theta \cdot \sigma_{22}) \times \cos\theta$$

1 element - $1C_1$

$$(\cos^2\theta \cdot \sigma_{11} + \sin^2\theta \cdot \sigma_{22} + \sin\theta \cdot \cos\theta \cdot \sigma_{21} + \sin\theta \cdot \cos\theta \cdot \sigma_{12})$$

element - $2C_2$

$$-\sin^2\theta \cdot \sigma_{21} + \cos^2\theta \cdot \sigma_{12} - \sin\theta \cdot \cos\theta \cdot \sigma_{11} + \sin\theta \cdot \cos\theta \cdot \sigma_{22}$$

2nd row
element - $3C_1$

$$-\sin^2\theta \cdot \sigma_{21} - \sin\theta \cos\theta \cdot \sigma_{11} + \cos^2\theta \cdot \sigma_{12} + \sin\theta \cos\theta \cdot \sigma_{22}$$

element - $4C_2$

$$\sin^2\theta \cdot \sigma_{11} + -\sin\theta \cos\theta \cdot \sigma_{21} + \cos^2\theta \cdot \sigma_{22} - \sin\theta \cos\theta \cdot \sigma_{12}$$

Note - Replacing xx with xx
and yy with yy

or we can say 1 x - Replaced with x &
2 y - Replaced with y for
better understanding -

In 2D, the transformation eqs are from the Complex (Resultant) matrix -

$$\sigma'_{xx} = \sigma_{xx} \cos^2 \theta + \sigma_{yy} \sin^2 \theta + 2\tau_{xy} \sin \theta \cos \theta$$

$$\sigma'_{yy} = \sigma_{xx} \sin^2 \theta + \sigma_{yy} \cos^2 \theta - 2\tau_{xy} \sin \theta \cos \theta$$

$$\tau'_{xy} = (\sigma_{yy} - \sigma_{xx}) \sin \theta \cos \theta + \tau_{xy} (\cos^2 \theta - \sin^2 \theta)$$

In 2-D, the principal stress orientation, θ_p , can be

computed by setting $\tau'_{xy} = 0$ in the above shear equation and solving for θ to get θ_p , the principal stress angle.

$$0 = (\sigma_{yy} - \sigma_{xx}) \sin \theta_p \cos \theta_p + \tau_{xy} (\cos^2 \theta_p - \sin^2 \theta_p)$$

This gives

$$\tan(2\theta_p)$$

The transformation Q is

$$Q = \begin{bmatrix} \cos \theta_p & \sin \theta_p \\ -\sin \theta_p & \cos \theta_p \end{bmatrix}$$

interest inserting this value θ for θ_p
the equations for the normal stresses gives
the principal values they are written as

σ_{max} and σ_{min} or alternatively
as σ_1 and σ_2

$$\sigma_{max}, \sigma_{min} = \frac{\sigma_{xx} + \sigma_{yy}}{2} \pm \sqrt{\left(\frac{\sigma_{xx} - \sigma_{yy}}{2}\right)^2 + \tau_{xy}^2}$$

They could also be obtained by

$$\sigma' = Q \cdot \sigma \cdot Q^T \text{ with } Q \text{ based on } \theta_p$$

• Principal stress Notation -

- principal stresses can be written as σ_1, σ_2 and σ_3 .

only one subscript is usually used in this
case to differentiate the principal stress
values from the normal stress

components σ_{11}, σ_{22} and σ_{33} .

Problem-05 - Solution -

Given,

$$[\sigma_{ij}] = \begin{bmatrix} 5/2 & -1/2 & 0 \\ -1/2 & 5/2 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

The principal values are the solution to the characteristic eqn -

$$\begin{vmatrix} 5/2 - \lambda & -1/2 & 0 \\ -1/2 & 5/2 - \lambda & 0 \\ 0 & 0 & 1 - \lambda \end{vmatrix}$$

$$= \left(\frac{5}{2} - \lambda \right) \left(\frac{5}{2} - \lambda \right) (1 - \lambda) - 0 - \frac{1}{2}$$

$$= (1 - \lambda) \left\{ \left(\frac{5}{2} - \lambda \right)^2 - \frac{1}{4} \right\}$$

$$= (1 - \lambda) \left(\frac{25}{4} + \lambda^2 - 5\lambda - \frac{1}{4} \right)$$

$$= (1 - \lambda) (\lambda^2 - 5\lambda - 6)$$

$$= \cancel{(1 - \lambda)} \cancel{(\lambda - 3)} \cancel{(\lambda - 2)}$$

$$= (1 - \lambda) (\lambda - 6) (\lambda + 1) \times$$

Three principal values are -

~~Λ_{20}~~ ~~Λ_{30}~~
 ~~Λ_1 or $\sigma_1 = 1$~~ , ~~$\sigma_2 = 6$~~ ~~$\sigma_3 = -1$~~

σ_1 or $\Lambda_1 = 6$

σ_3 or $\Lambda_3 = -1$

and σ_2 or $\Lambda_2 = 1$

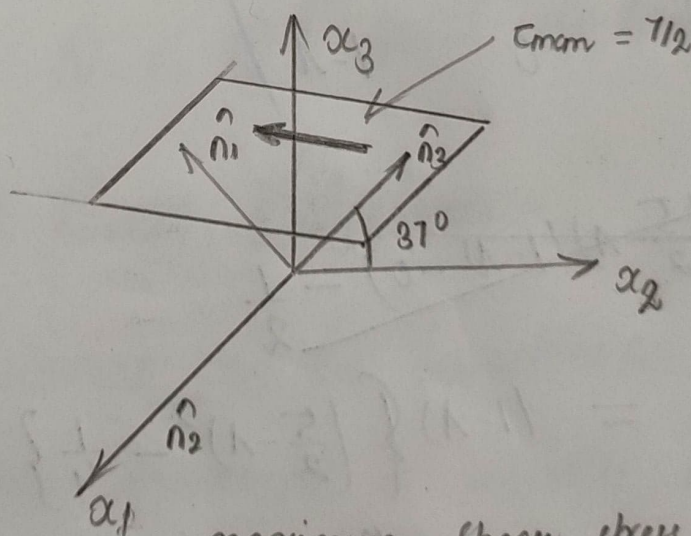
and so the maximum

shear stress is

$$\tau_{max} = \frac{1}{2} (\sigma_1 - \sigma_3) =$$

$$= \frac{1}{2} (6 + 1)$$

$$\tau_{max} = 7/2$$



maximum shear stress direction