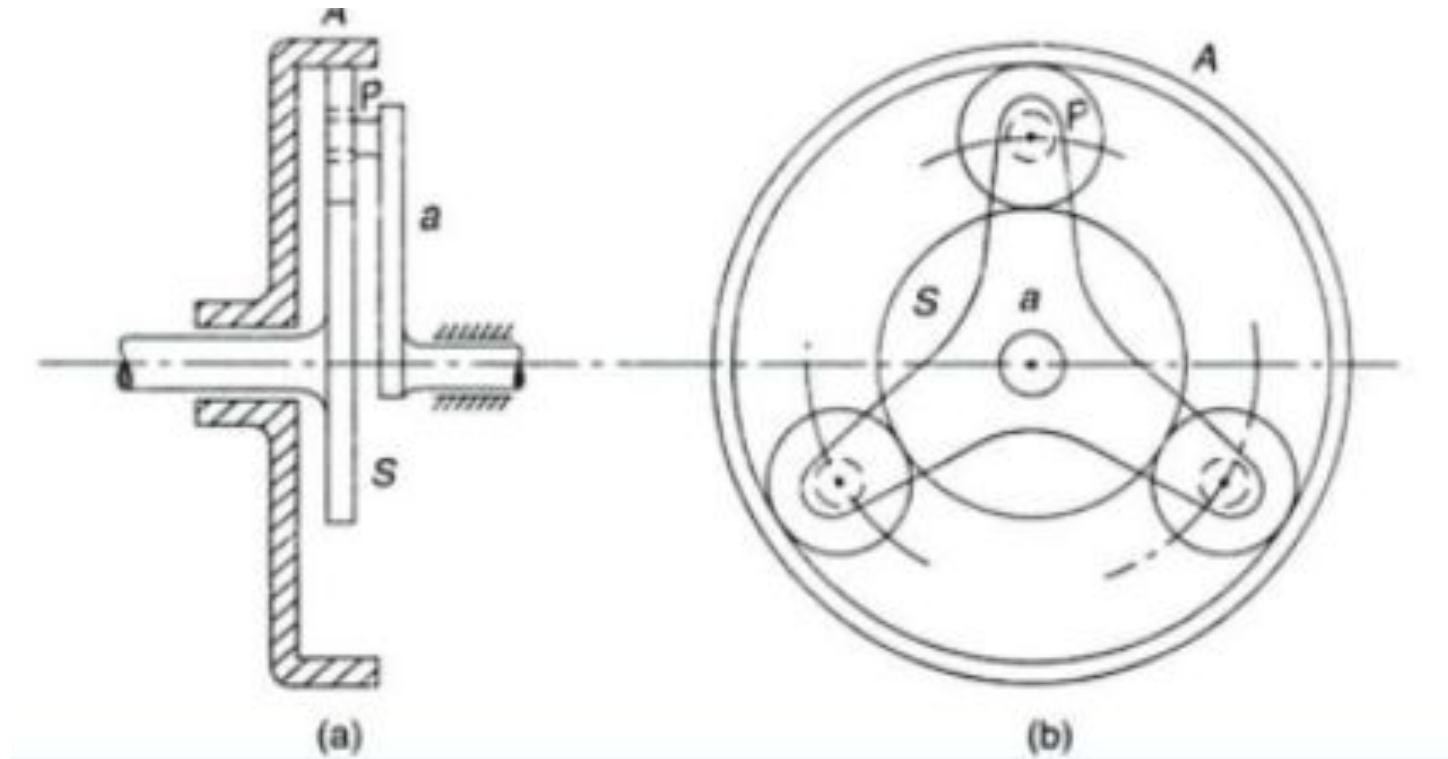


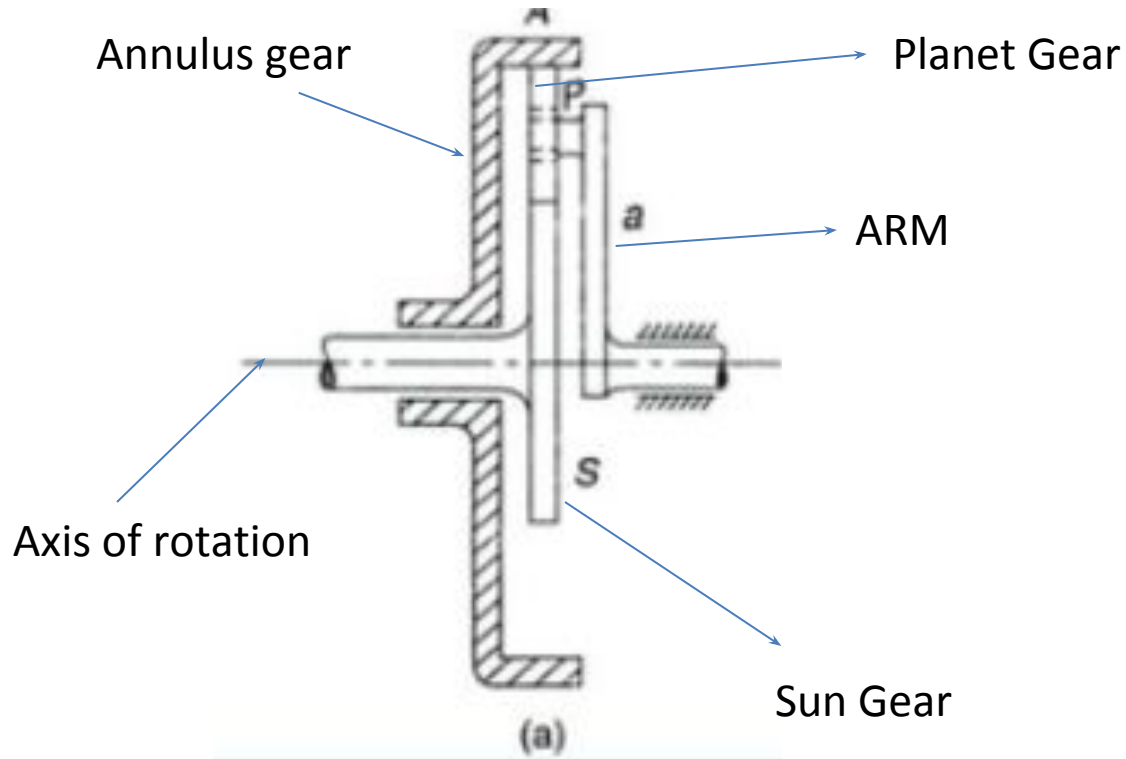
MEE-S301

Epicyclic gear train

Q1. The annulus A in the gear shown in the fig 1. rotates at 300 rpm about the axis of the fixed wheel S which has 80 teeth. The three-armed spider (only one arm a is shown in figure) is driven at 180 rpm. Determine the number of teeth required on the wheel P.



First identify gear name



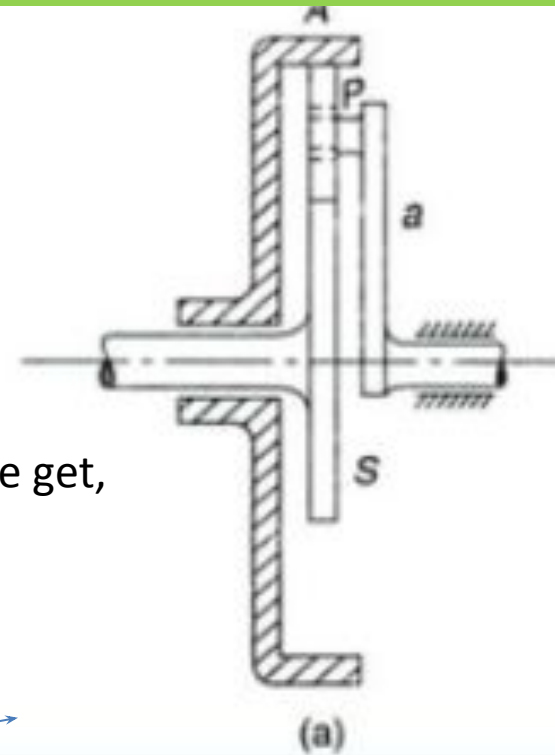
Sun Gear : Sun gear rotates about fixed axis of rotation.

Arm : Arm rotates about fixed axis of rotation.

Planet: Axis of rotation of planet is not fixed.



	Sun gear (S)	Planet gear (P)	Annulus (A)	ARM (a)
Speed (N)	$N_S = 0$	$N_P = ??$	$N_A = 300 \text{ rpm}$	$N_{\text{arm}} = 180 \text{ rpm}$
No of teeth	$Z_S = 80$	$Z_P = ??$	$Z_A = ??$	-



- Sun gear, planet gear, arm constitute external epicyclic gear train

By using relative velocity method,

$$\frac{N_S - N_{\text{Arm}}}{N_P - N_{\text{Arm}}} = - \frac{Z_P}{Z_S} \quad [N_S = 0, N_{\text{ARM}} = 180 \text{ rpm}]$$

$$\frac{0 - 180}{N_P - 180} = - \frac{Z_P}{80} \quad \text{---- (1)}$$

- Internal gear A, planet gear P, arm constitute internal epicyclic gear train

By using relative velocity method,

$$\frac{N_A - N_{\text{Arm}}}{N_P - N_{\text{Arm}}} = + \frac{Z_P}{Z_A} \quad [N_A = 300 \text{ rpm}, N_{\text{ARM}} = 180 \text{ rpm}]$$

$$\frac{300 - 180}{N_P - 180} = + \frac{Z_P}{Z_A} \quad \text{---- (2)}$$

Divide (1) and (2) we get,

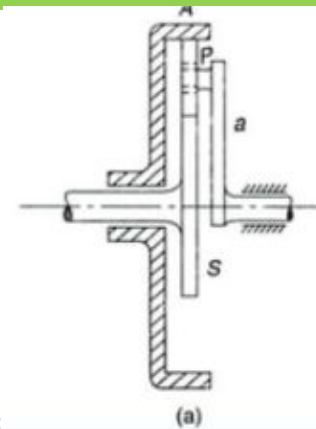
$$\frac{\left(\frac{0 - 180}{N_P - 180} \right)}{\left(\frac{300 - 180}{N_P - 180} \right)} = \frac{- \frac{Z_P}{80}}{\frac{Z_P}{Z_A}}$$

$$\frac{180}{120} = \frac{Z_A}{80}$$

$$Z_A = \frac{3}{2} \times 80 = 120$$

Number of teeth on gear A is equal to 120.

	Sun gear (S)	Planet gear (P)	Annulus (A)	ARM (a)
Speed (N)	$N_s = 0$	$N_p = 900 \text{ rpm}$	$N_A = 300 \text{ rpm}$	$N_{arm} = 180 \text{ rpm}$
No of teeth	$Z_s = 80$	$Z_p = 20$	$Z_A = 120$	-



Since gear S is mesh with gear P.
Therefore module of gear S is equal to module of gear P

$$\text{Module of gear S} = \frac{2r_s}{Z_S}$$

$$\text{Module of gear P} = \frac{2r_p}{Z_P}$$

Similarly gear P is mesh with internal gear A. Therefore module of gear P is equal to module of gear A.

$$\text{Module of gear P} = \frac{2r_p}{Z_P}$$

$$\text{Module of gear A} = \frac{2r_A}{Z_A}$$

Module of gear S = Module of gear P =
Module of gear A

$$\frac{2r_s}{Z_S} = \frac{2r_p}{Z_P} = \frac{2r_A}{Z_A} = m$$

$$\begin{aligned} \text{Also } r_A &= r_s + 2r_p \\ Z_A &= Z_S + 2Z_P \\ 120 &= 80 + 2 \times Z_P \end{aligned}$$

$$Z_P = 20$$

$$N_P = 900 \text{ rpm}$$

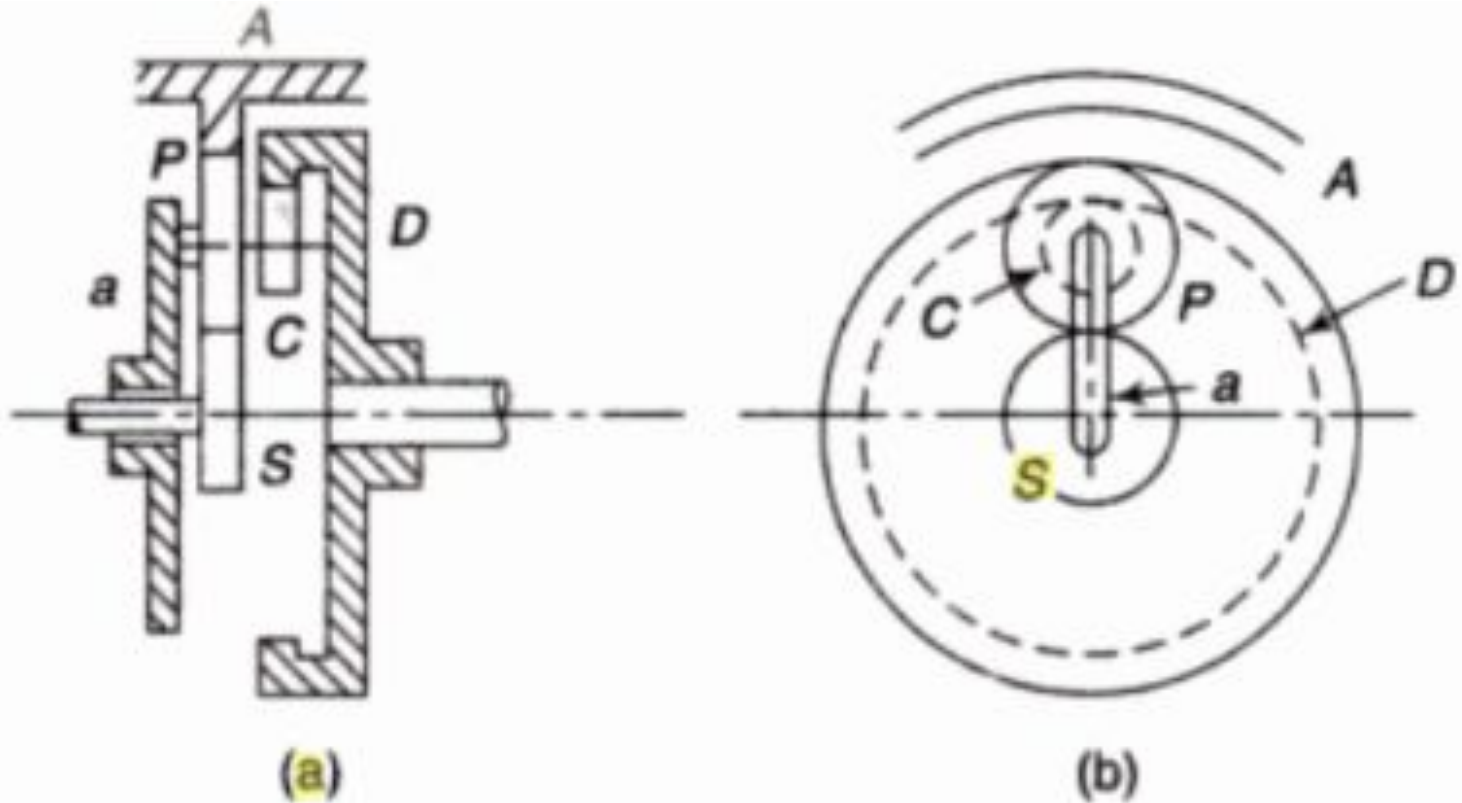
$$\frac{0 - 180}{N_P - 180} = -\frac{Z_P}{80}$$

From this equation we can find speed of planet gear

Problem-2

Epicyclic gear train

Q2. In a reduction gear shown in Fig -2 , the input S has 24 teeth. P and C constitute a compound planet a compound planet having 30 and 18 teeth respectively. If all the gears are of the same pitch, find the ratio of the reduction of gear. Assume A to be fixed



First identify the sun gear, planet gear and arm

	Sun gear (S)	Planet gear (P)	Arm (a)	Ring gear -A	Planet gear -C	Internal gear D
Speed (N) rpm				0		
No of teeth (Z)	24	30	-	84	18	

➤ Gear A is fixed , $N_A = 0$ rpm

➤ Gear S , Gear P and arm constitute epicyclic gear train.

$$\frac{N_S - N_{Arm}}{N_P - N_{Arm}} = - \frac{Z_P}{Z_S}$$

$$\frac{N_S - N_{Arm}}{N_P - N_{Arm}} = - \frac{30}{24} \text{ ---- (1)}$$

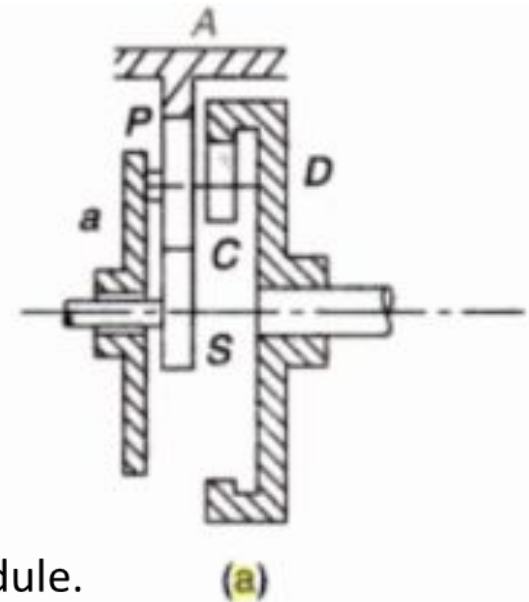
From figure,

$$r_A = r_s + 2r_p$$

Since all gear are have same pitch it means it have also same module.

$$Z_A = Z_s + 2Z_p$$

$$Z_A = 24 + 2 \times 30 = \mathbf{84}$$



First identify the sun gear, planet gear and arm

	Sun gear (S)	Planet gear (P)	Arm (a)	Ring gear -A	Planet gear -C	Internal gear D
Speed (N) rpm				0		
No of teeth (Z)	24	30	-	84	18	72

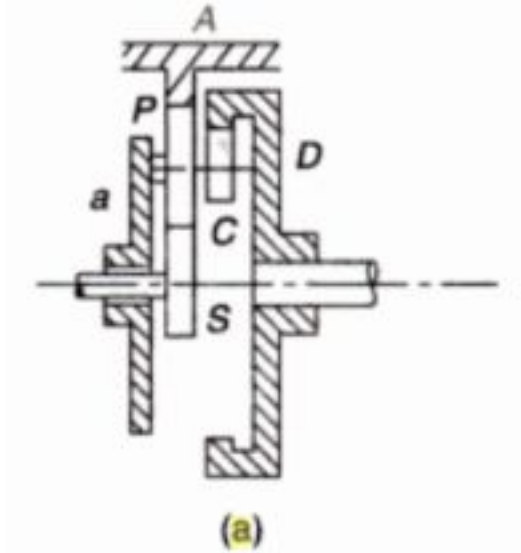
From figure,

$$r_D = r_s + r_p + r_c$$

Since all gear are have same pitch it means it have also same module.

$$Z_D = Z_s + Z_p + Z_c$$

$$Z_D = 24 + 30 + 18 = 72$$



72

72

➤ Gear P, Gear – A , arm constitute internal epicyclic gear train.

$$\frac{N_A - N_{Arm}}{N_P - N_{Arm}} = + \frac{Z_P}{Z_A}$$

$$\frac{0 - N_{Arm}}{N_P - N_{Arm}} = \frac{30}{84}$$

$$-84N_{Arm} = 30N_P - 30N_{Arm}$$

$$-54N_{Arm} = 30N_P$$

$$N_P = -\frac{54}{30}N_{Arm}$$

First identify the sun gear, planet gear and arm

	Sun gear (S)	Planet gear (P)	Arm (a)	Ring gear -A	Planet gear -C	Internal gear D
Speed (N) rpm				0		
No of teeth (Z)	24	30	-	84	18	72

By equation (1)

$$\frac{N_S - N_{Arm}}{N_P - N_{Arm}} = -\frac{30}{24} \text{ ---- (1)}$$

$$\frac{N_S - N_{Arm}}{-\frac{54}{30}N_{Arm} - N_{Arm}} = -\frac{30}{24}$$

$$N_S = N_{arm} - \frac{30}{24} \times -\frac{84}{30} N_{Arm}$$

$$N_S = N_{Arm} + \frac{84}{24} N_{Arm} = \frac{108}{24} N_{Arm}$$

$$N_S = \frac{108}{24} N_{Arm}$$

Since planet gear P and gear C mount on same shaft. Therefore both have same speed.

$$N_P = N_C$$

Gear D, gear C and arm constitute internal epicyclic gear train

$$\frac{N_C - N_{Arm}}{N_D - N_{Arm}} = +\frac{Z_D}{Z_C} = \frac{72}{18} = 4$$

$$N_D = \frac{N_C - N_{Arm}}{4} + N_{arm} = \frac{N_C}{4} + \frac{3}{4} N_{arm}$$

First identify the sun gear, planet gear and arm

	Sun gear (S)	Planet gear (P)	Arm (a)	Ring gear -A	Planet gear -C	Internal gear D
Speed (N) rpm				0		
No of teeth (Z)	24	30	-	84	18	72

$$N_D = \frac{N_c}{4} + \frac{3}{4} N_{arm}$$

$$N_D = -\frac{54}{120} N_{arm} + \frac{3}{4} N_{arm}$$

$$N_D = \frac{-54 + 3 \times 30}{120} N_{arm}$$

$$N_D = \frac{36}{120} N_{arm}$$

Since planet gear P and gear C mount on same shaft. Therefore both have same speed.

$$N_P = N_C$$

$$\text{Speed reduction} = \frac{\text{Speed of input shaft}}{\text{Speed of output shaft}}$$

$$= \frac{N_S}{N_D} = \frac{\frac{108}{24} N_{arm}}{\frac{36}{120} N_{arm}} = 15$$

Ans: 15 (it means speed of output shaft reduced
By a factor of 15)